

Computational Models for the Growth of Closed Bacterial Cell Envelopes

Paul Schulze 2023-01-19

Outline



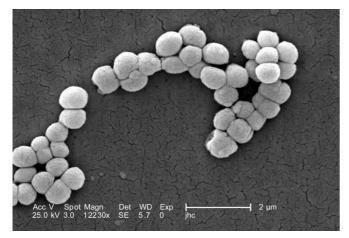
- Introduction
- Spring-based model
 - Idea behind the model
 - Results
- Finite-element-based model
 - Idea behind the model
 - Results
- Summary
- Outlook



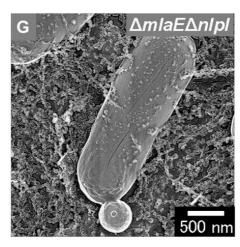
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Shapes of bacteria

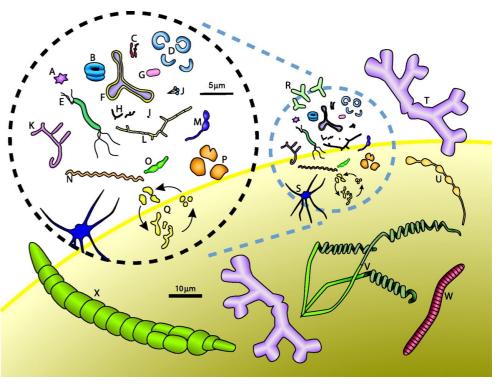




Carr, J. H. (2007) Public Health Image Library (PHIL), CDC



Ojima, Y. (2021) Microbial Physiology and Metabolism

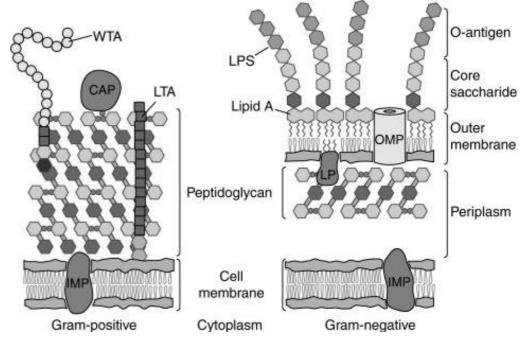


Young, K. D. (2006) Microbiology and molecular biology reviews

ТИП

The cell wall determines the shape

- Composed of Peptodoglycan
- Protects against osmotic pressure
- Two classes of bacteria
 - Gram-positive: Thick cell wall
 - Gram-negative: Thin cell wall



Silhavy, T. J. (2010) in Cold Spring Harbor perspectives in biology



Shape preservation in bacteria

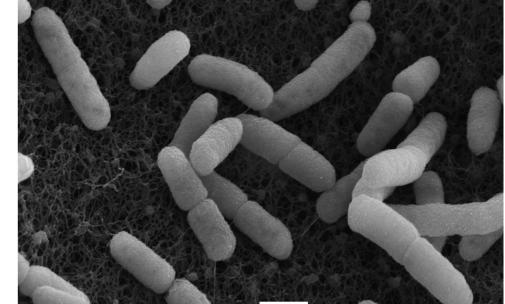
Shape is consistent through generations

10 min

20 min

30 min

- Shape is recovered after disturbance
- Shape changes in response to the environment





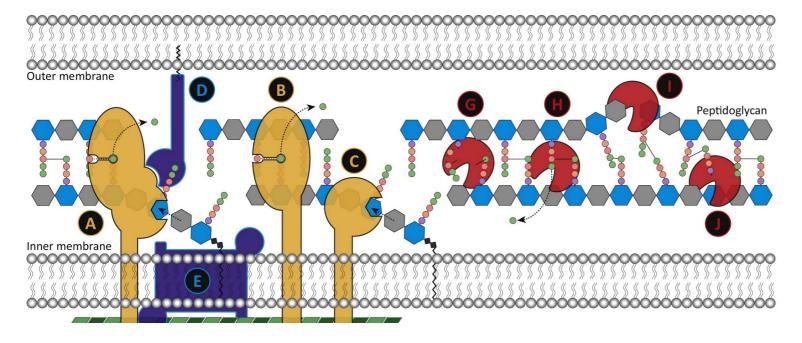
Gudrun Holland, Michael Laue/RKI

Reproduced from: Felix Wong et al. (2017) Nature microbiology

2 min



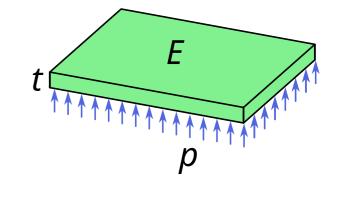
Peptidoglycan remodelling

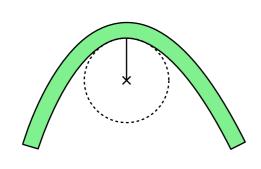


- PG synthases assemble the PG network
- PG hydrolases modify existing PG
- Guided by mechanical / geometrical cues?

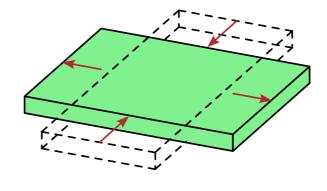
Growth model for the bacterial shell

- Mechanical properties of the cell wall
- Turgor pressure
- Growth rates are based on local cues





Curvature



Strain



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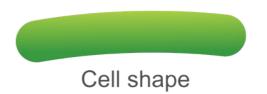
ПЛ

Spring-based model of the shell

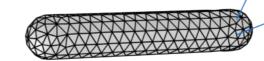
- Linear elastic continuum
 - Young's modulus E
 - Thickness *t*
- Spring network
 - Spring stiffness k_s
 - Bending stiffness k_b

Mapping:

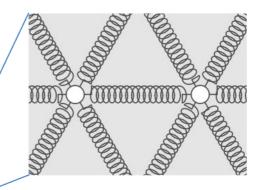
$$E, t \rightarrow k_s, k_b$$







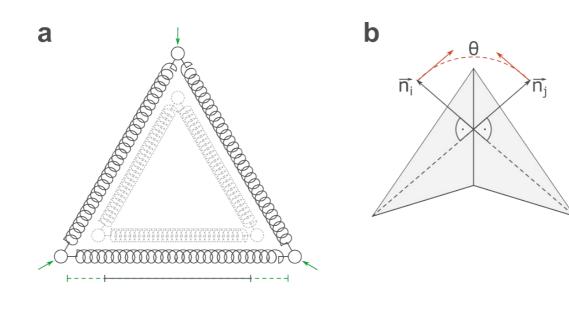
Triangulated surface



Linear elastic surface modelled as a network of Hookean springs.







Spring Energy

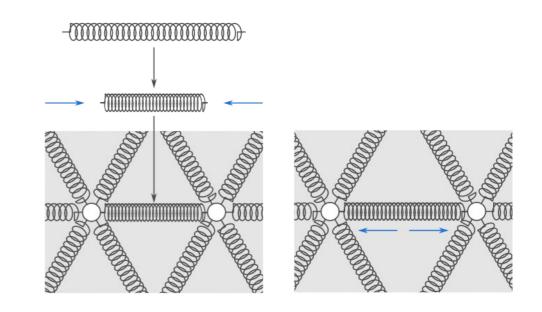
Bending Energy Pressure Energy

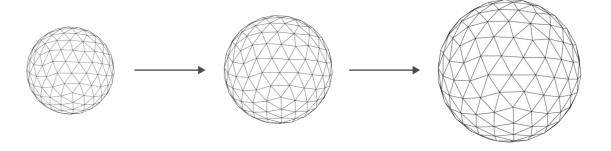
$$E = \sum_{e}^{edges} \frac{k_s}{2} (l_e - l_{0,e})^2 + \sum_{\{i,j\}(adj.)}^{triangles} k_b (1 - \vec{n}_i \vec{n}_j) - pV$$





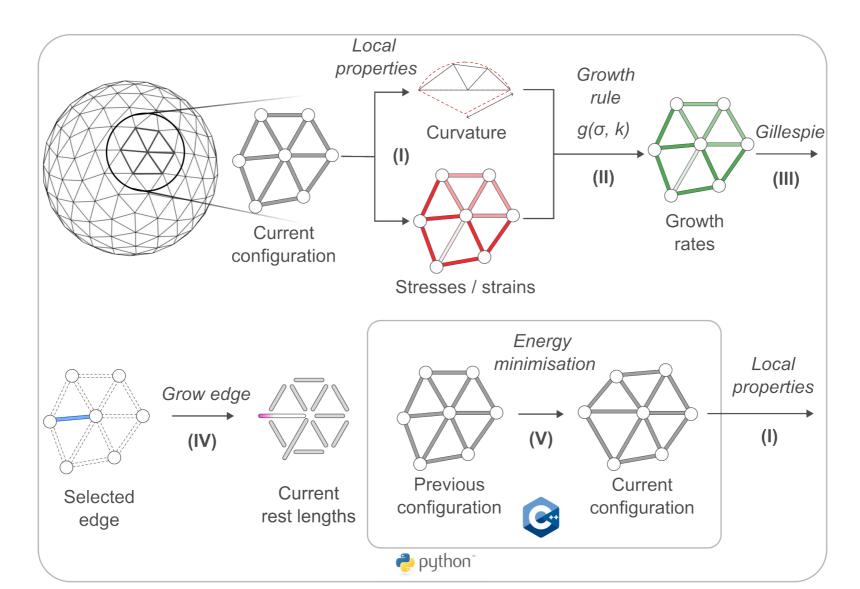
- Increasing rest lengths $I_{\rm e,0}$
- Mesh topology is consistent between growth steps









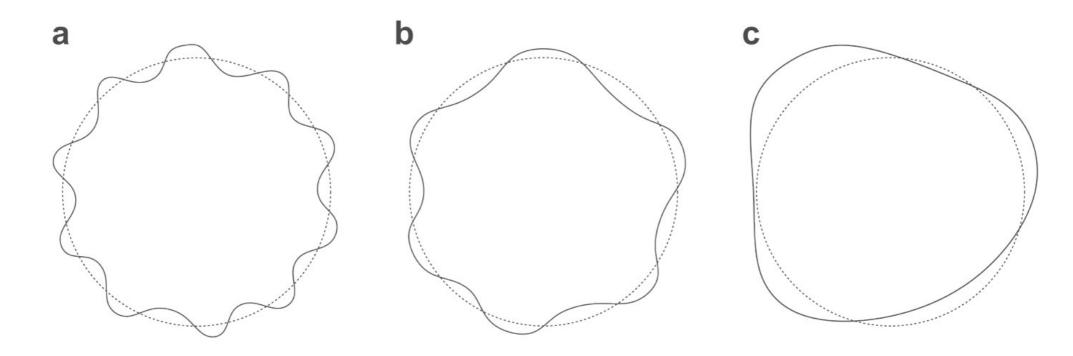




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Observables



Roughness

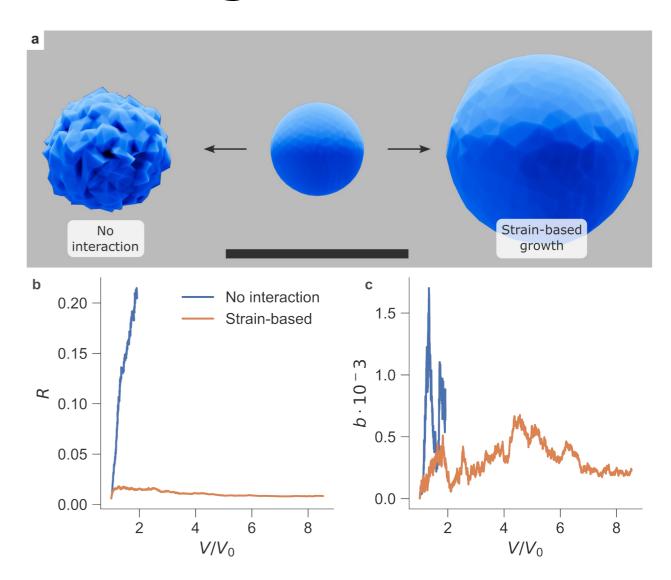
Asphericity



Strain-based growth rates:

$$\varepsilon_e = (l_e - l_{e,0})/l_{e,0}$$

$$\lambda_e = -\lambda_0 + \lambda_1 \varepsilon_e$$





Strain-based growth rates:

$$\varepsilon_e = (l_e - l_{e,0})/l_{e,0}$$

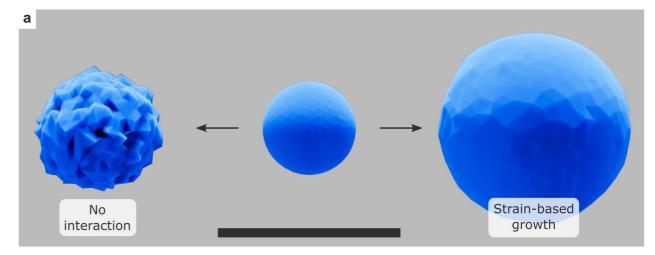
$$\lambda_e = -\lambda_0 + \lambda_1 \varepsilon_e$$

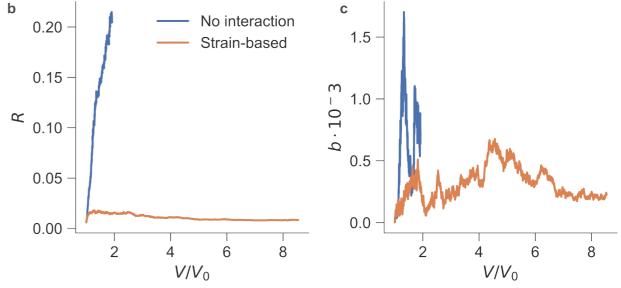
Random

- → Spherical shape
- → Surface roughness

Strain-based

- → Spherical shape
- → Smooth surface

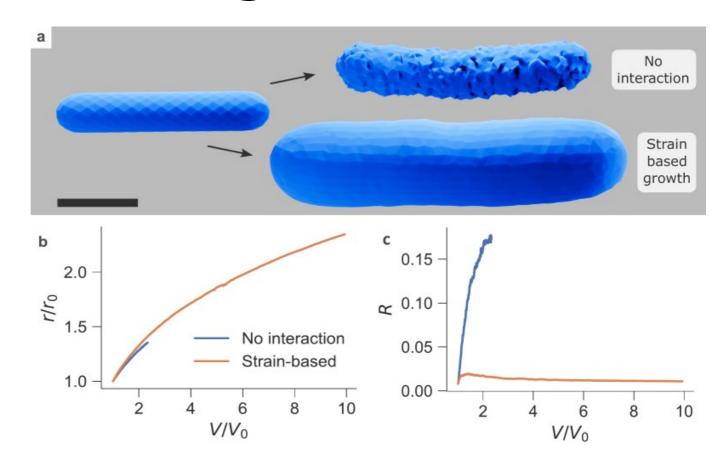






Strain-based growth rates:

$$\lambda_e = -\lambda_0 + \lambda_1 \varepsilon_e$$





Strain-based growth rates:

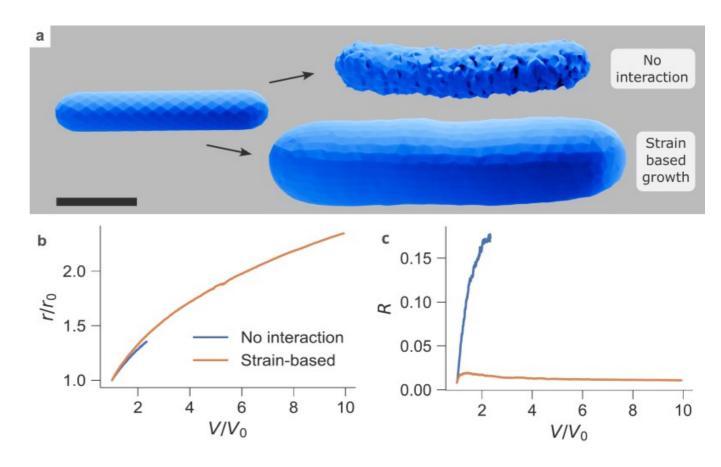
$$\lambda_e = -\lambda_0 + \lambda_1 \varepsilon_e$$

Random growth

- → Cylindrical shape
- → Surface roughness

Strain-based growth

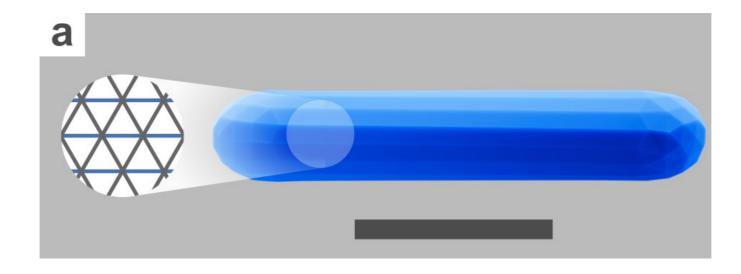
- → Cylindrical shape
- → Smooth surface

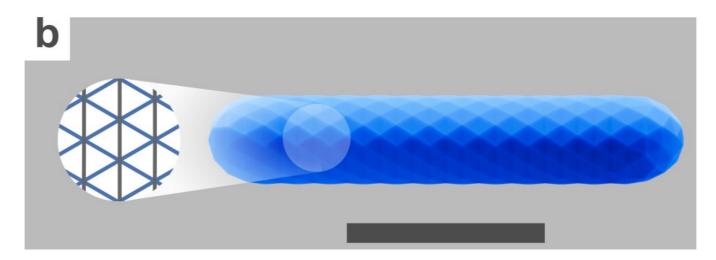


→ Radius is not conserved



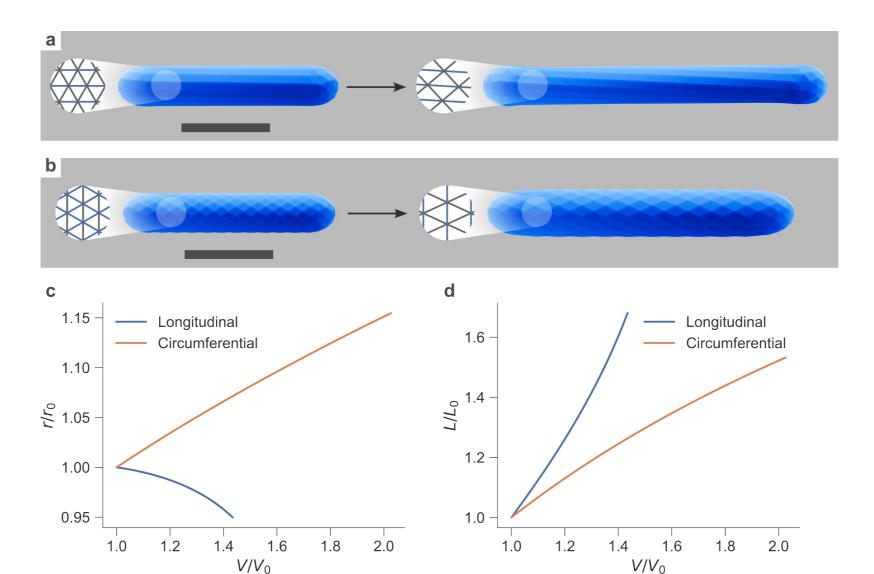
Directed growth: Mesh alignment







Directed growth: Mesh alignment

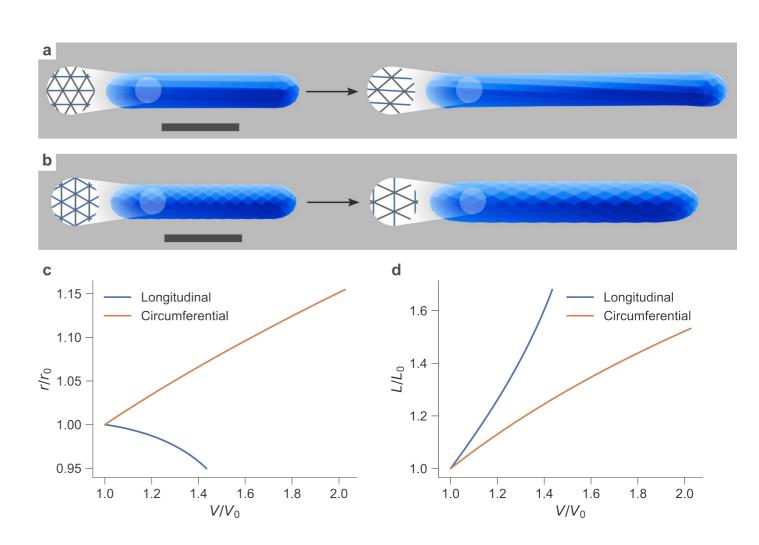




Spring-based model: Limitations

- Dependence on mesh alignment
- Acute angles in triangles
 - → Not linear elastic
- Hyperelasticity

→ Alternative model



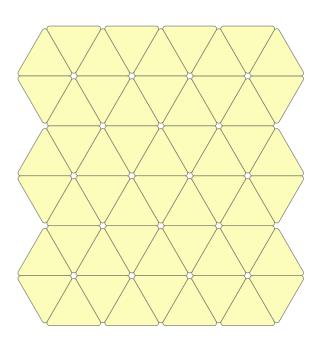


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Finite element method

- Method to solve partial differential equations (PDEs)
- Solve for deformations of complex geometries under outside forces
- Surface is divided into elements
- Solution is evaluated at nodes
- Interpolated for a full solution







Strain for triangular element:

$$\vec{\varepsilon} = [B] \vec{u}_{\text{element}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ 2\varepsilon_{12} \end{bmatrix} = \begin{bmatrix} \frac{\partial N_a}{\partial x_1} & 0 & \frac{\partial N_b}{\partial x_1} & 0 & \frac{\partial N_c}{\partial x_1} & 0 \\ 0 & \frac{\partial N_a}{\partial x_2} & 0 & \frac{\partial N_b}{\partial x_2} & 0 & \frac{\partial N_c}{\partial x_2} \\ \frac{\partial N_a}{\partial x_1} & \frac{\partial N_a}{\partial x_2} & \frac{\partial N_b}{\partial x_1} & \frac{\partial N_b}{\partial x_2} & \frac{\partial N_c}{\partial x_1} & \frac{\partial N_c}{\partial x_2} \end{bmatrix} \begin{bmatrix} u_1^{(a)} \\ u_2^{(a)} \\ u_2^{(b)} \\ u_1^{(b)} \\ u_2^{(c)} \\ u_1^{(c)} \\ u_2^{(c)} \end{bmatrix}$$





Strain for triangular element:

$$\vec{\varepsilon} = [B] \vec{u}_{\text{element}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ 2\varepsilon_{12} \end{bmatrix} = \begin{bmatrix} \frac{\partial N_a}{\partial x_1} & 0 & \frac{\partial N_b}{\partial x_1} & 0 & \frac{\partial N_c}{\partial x_1} & 0 \\ 0 & \frac{\partial N_a}{\partial x_2} & 0 & \frac{\partial N_b}{\partial x_2} & 0 & \frac{\partial N_c}{\partial x_2} \\ \frac{\partial N_a}{\partial x_1} & \frac{\partial N_a}{\partial x_2} & \frac{\partial N_b}{\partial x_1} & \frac{\partial N_b}{\partial x_2} & \frac{\partial N_c}{\partial x_1} & \frac{\partial N_c}{\partial x_2} \end{bmatrix} \begin{bmatrix} u_1^{(a)} \\ u_2^{(b)} \\ u_1^{(b)} \\ u_2^{(c)} \\ u_1^{(c)} \\ u_2^{(c)} \end{bmatrix}$$

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \end{bmatrix} = [D] \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \end{bmatrix} = \frac{E}{(1-v)^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \end{bmatrix}$$

Stress-strain relation:

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = [D] \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{12} \end{bmatrix} = \frac{E}{(1-\nu)^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & (1-\nu)/2 \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{12} \end{bmatrix}$$





Strain for triangular element:

$$\vec{\varepsilon} = [B] \vec{u}_{\text{element}} = \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ 2\varepsilon_{12} \end{bmatrix} = \begin{bmatrix} \frac{\partial N_a}{\partial x_1} & 0 & \frac{\partial N_b}{\partial x_1} & 0 & \frac{\partial N_c}{\partial x_2} & 0 \\ 0 & \frac{\partial N_a}{\partial x_2} & 0 & \frac{\partial N_b}{\partial x_2} & 0 & \frac{\partial N_c}{\partial x_2} \\ \frac{\partial N_a}{\partial x_1} & \frac{\partial N_a}{\partial x_2} & \frac{\partial N_b}{\partial x_1} & \frac{\partial N_b}{\partial x_2} & \frac{\partial N_c}{\partial x_1} & \frac{\partial N_c}{\partial x_2} \end{bmatrix} \begin{bmatrix} u_1^{(a)} \\ u_2^{(a)} \\ u_2^{(b)} \\ u_1^{(b)} \\ u_2^{(c)} \\ u_1^{(c)} \\ u_2^{(c)} \end{bmatrix}$$

Stress-strain relation:

$$\begin{bmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{12} \end{bmatrix} = [D] \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{12} \end{bmatrix} = \frac{E}{(1-\nu)^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & (1-\nu)/2 \end{bmatrix} \begin{bmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{12} \end{bmatrix}$$

Strain energy:

$$U = \frac{1}{2} \vec{\varepsilon}^{\mathbf{T}} \vec{\sigma} = \frac{1}{2} \vec{\varepsilon}^{\mathbf{T}} [D] \vec{\varepsilon}.$$

Finite element method



$$\vec{\varepsilon} = [B]\vec{u}_{\text{element}} \quad U = \frac{1}{2}\vec{\varepsilon}^{\mathbf{T}}\vec{\sigma} = \frac{1}{2}\vec{\varepsilon}^{\mathbf{T}}[D]\vec{\varepsilon}.$$



Finite element method

$$\vec{\varepsilon} = [B]\vec{u}_{\text{element}} \quad U = \frac{1}{2}\vec{\varepsilon}^{\mathbf{T}}\vec{\sigma} = \frac{1}{2}\vec{\varepsilon}^{\mathbf{T}}[D]\vec{\varepsilon}.$$

$$W_{\text{element}} = \frac{1}{2} \vec{u}_{\text{element}}^{\mathbf{T}} (A_{\text{element}}[B]^{\mathbf{T}}[D][B]) \vec{u}_{\text{element}}$$





$$\vec{\varepsilon} = [B]\vec{u}_{\text{element}} \quad U = \frac{1}{2}\vec{\varepsilon}^{\mathbf{T}}\vec{\sigma} = \frac{1}{2}\vec{\varepsilon}^{\mathbf{T}}[D]\vec{\varepsilon}.$$

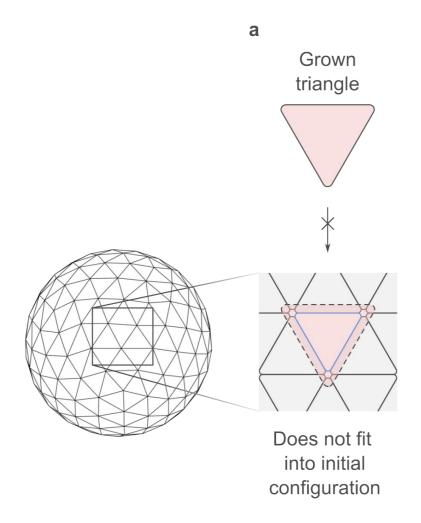
$$W_{\text{element}} = \frac{1}{2} \vec{u}_{\text{element}}^{\mathbf{T}} (A_{\text{element}}[B]^{\mathbf{T}}[D][B]) \vec{u}_{\text{element}}$$

$$W = \sum_{\text{elements}} W_{\text{element}} = \frac{1}{2} \sum_{\text{elements}} \vec{u}_{\text{element}}^{\mathbf{T}} K_{\text{element}} \vec{u}_{\text{element}}$$

Minimize *W* with respect to $u \rightarrow Deformations$

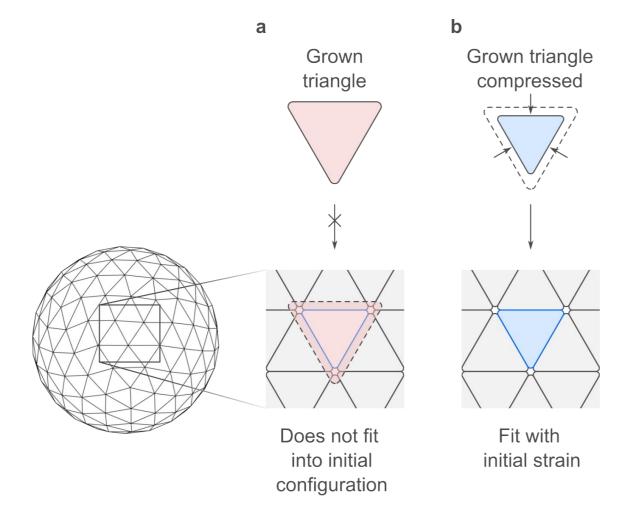


Finite-element-based Model



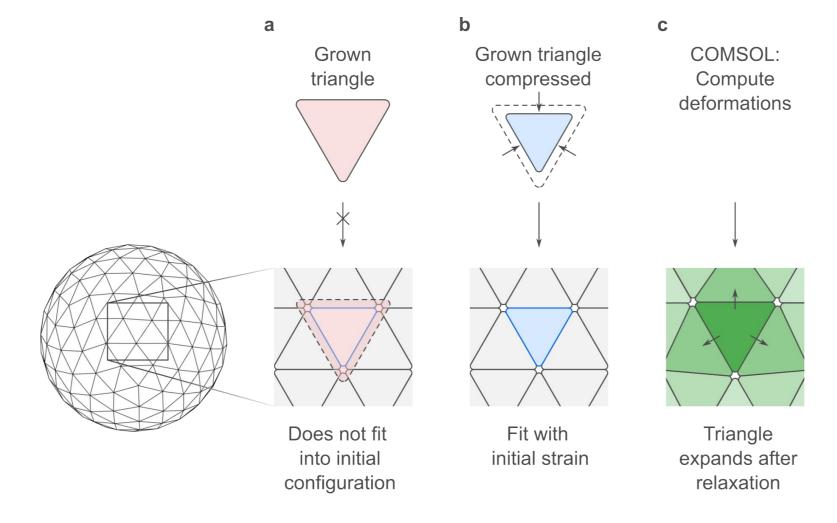






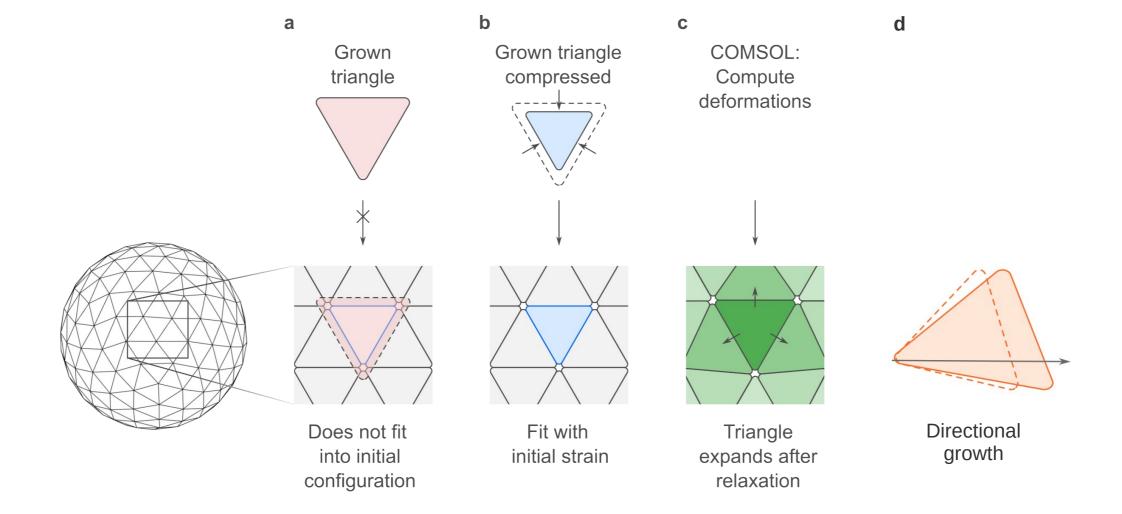






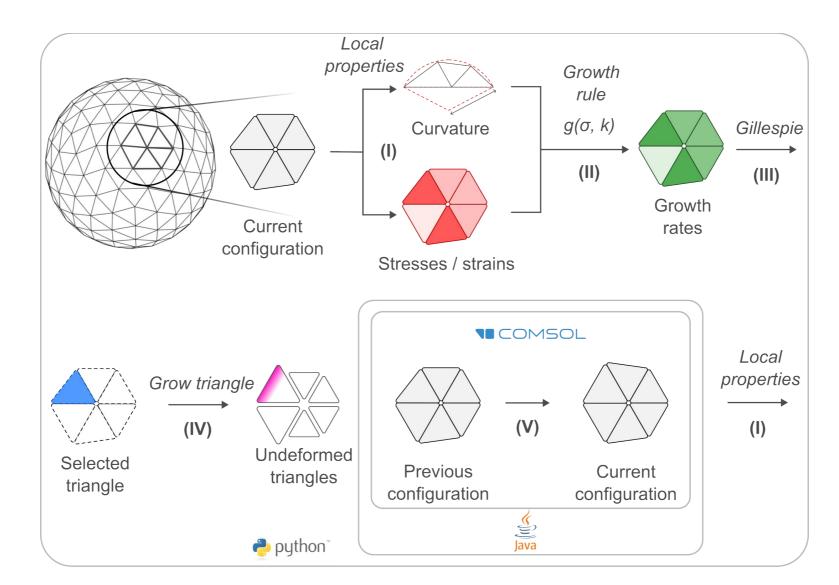
Finite-element-based Model









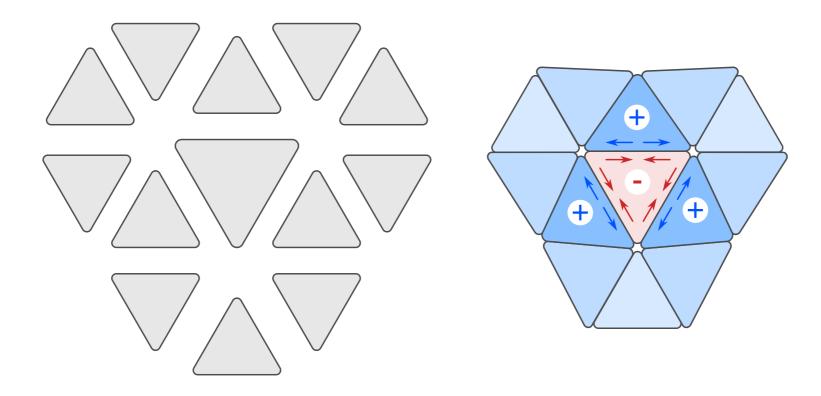




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Observable: Surface stresses





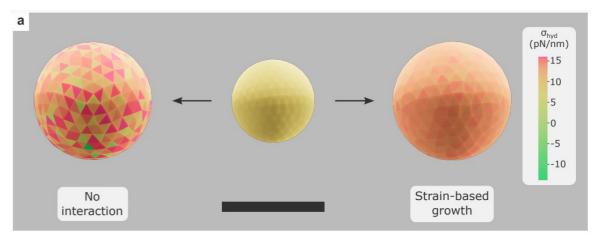
Size mismatch of elements

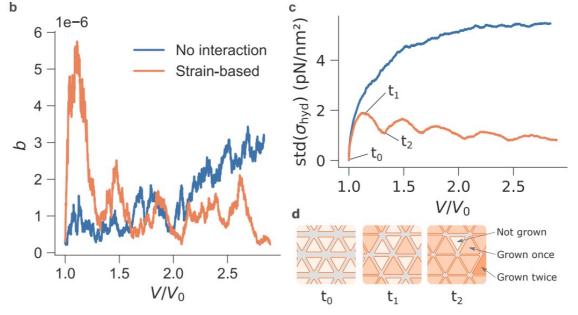
- → High compressive and tensile stresses
- → Affects mechanical stability



Strain-based growth rates:

$$\lambda^{(e)} = -\lambda_0 + \lambda_1 \varepsilon_h^{(e)}$$







Strain-based growth rates:

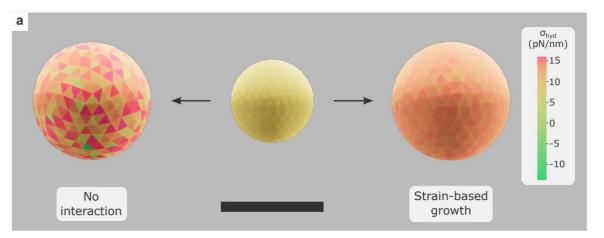
$$\lambda^{(e)} = -\lambda_0 + \lambda_1 \varepsilon_h^{(e)}$$

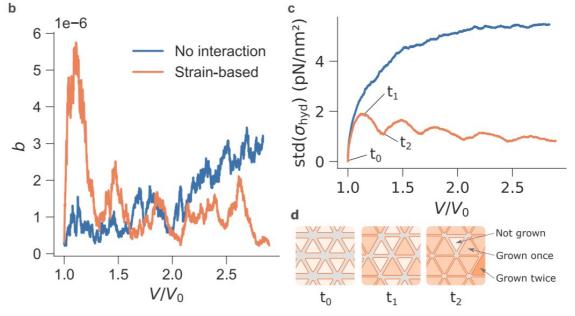
Random

- → Spherical shape
- → High surface stresses

Strain-based

- → Spherical shape
- → Moderate surface stresses

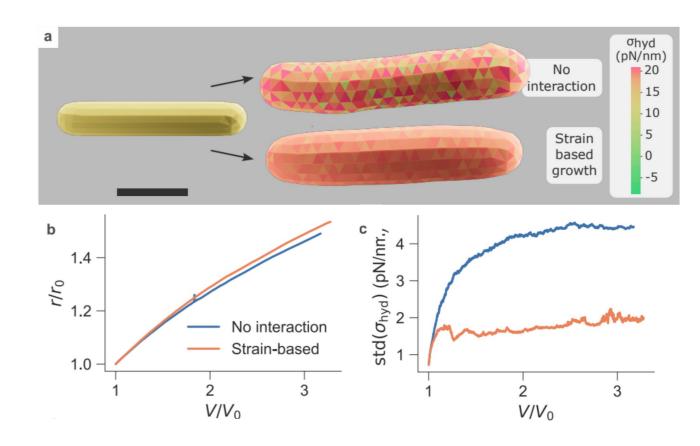






Strain-based growth rates:

$$\lambda^{(e)} = -\lambda_0 + \lambda_1 \varepsilon_h^{(e)}$$





Strain-based growth rates:

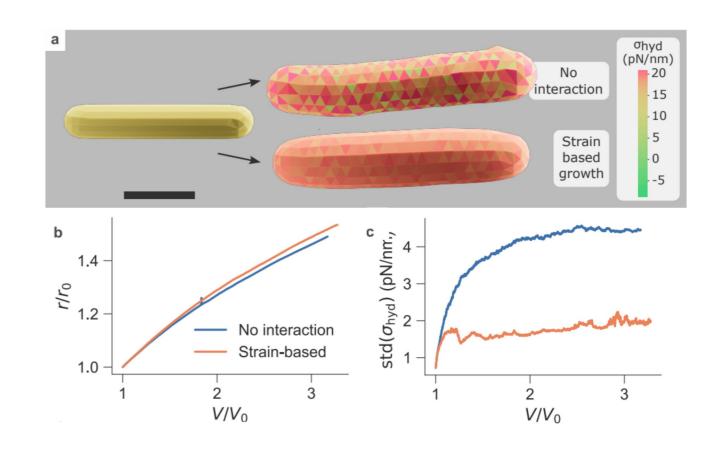
$$\lambda^{(e)} = -\lambda_0 + \lambda_1 \varepsilon_h^{(e)}$$

Random

- → Spherical shape
- → High stresses

Strain-based

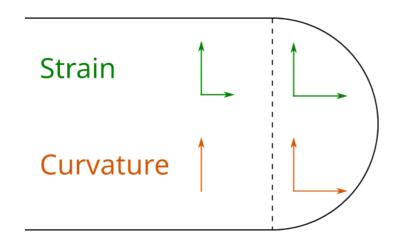
- → Spherical shape
- → Moderate stresses



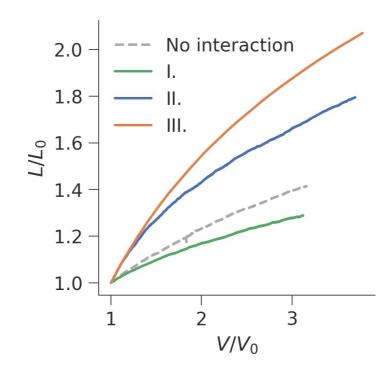
→ Radius is not conserved

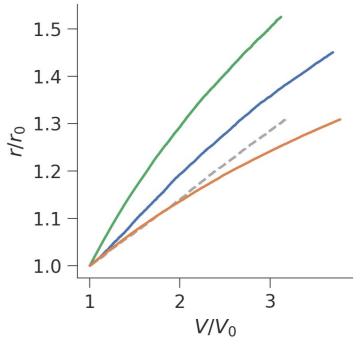
Directional Growth: Strain / Curvature





	Location	Direction
I.	Strain	Strain
II.	Curvature	Strain
III.	Curvature	Curvature





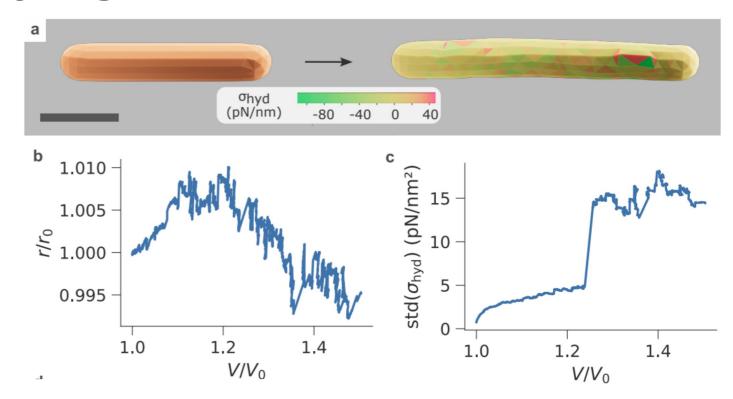


Correction mechanism

<u>Combination</u> longitudinal: +

circumferential:

- → Elongation
- → Radius is conserved
- → Problem: Large stresses





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Summary

- Implemented two physical models of the bacterial shell
- Limitations of the spring-based model
- Solved with finite-element-based model
- Strain-based growth improved the mechanical stability
- Importance of a correction mechanism



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Outlook



- Understanding high stresses
- Explore the parameter space
 - Gram-positive bacteria: Higher pressures, thicker cell wall
 - Pressure changes during growth
- Explore other geometries



Thank you



Appendix



$$Y = \frac{2}{\sqrt{3}}k_s,$$

$$Y = \frac{2}{\sqrt{3}}k_s,$$
$$\kappa_{\rm b} = \frac{\sqrt{3}}{2}k_{\rm b},$$

$$\kappa_b = \frac{Et^3}{12(1-\nu)}$$



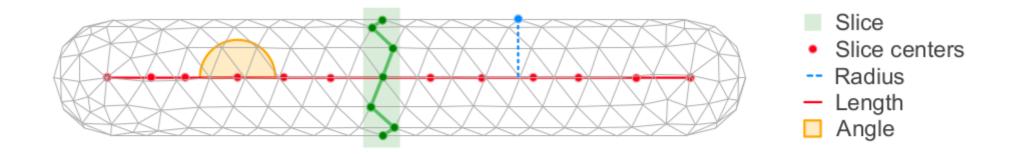
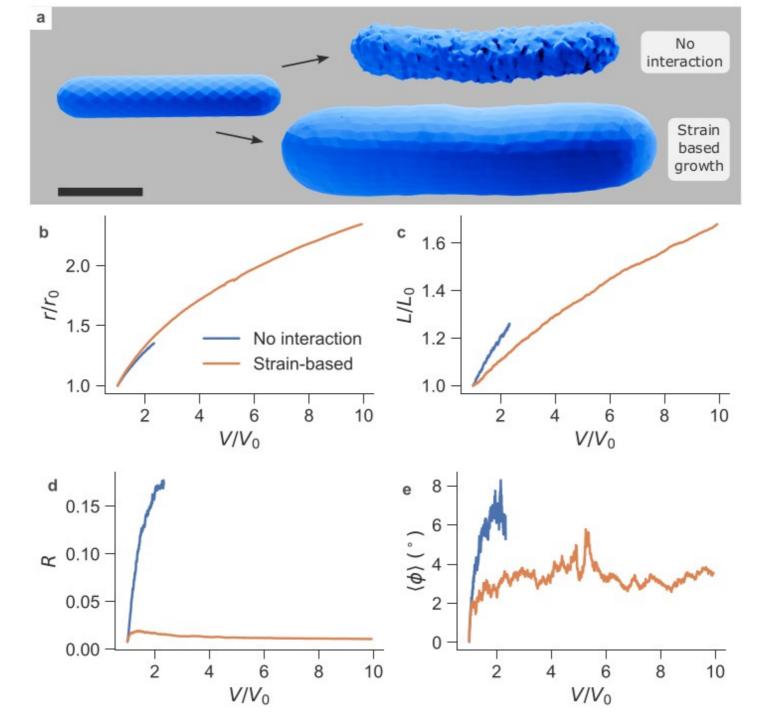


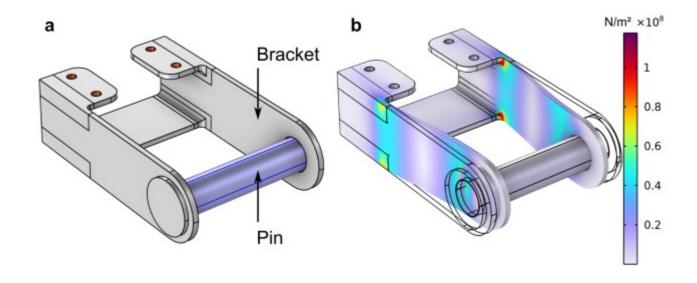
Figure 2.3.4: Computing length, radius and bending angle of the rod-shaped shell.

The shell is divided into slices (green) along its length. At initialization, each vertex is assigned a slice. The gravitational centers of the slices (red dots) can be computed at any point in the simulation. Radius, length and bending angles are computed from the line connecting the slice centers.





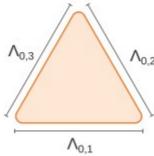


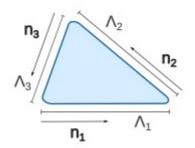


a Undeformed







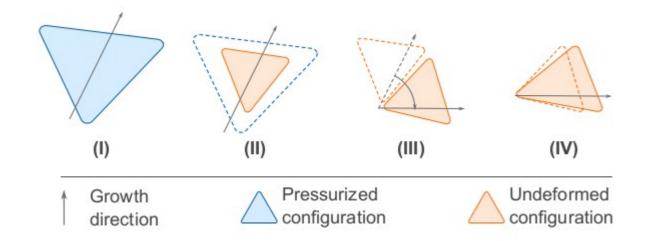


$$\varepsilon(\vec{n}^{(i)}) = \frac{\Lambda_i - \Lambda_{i,0}}{\Lambda_i}$$

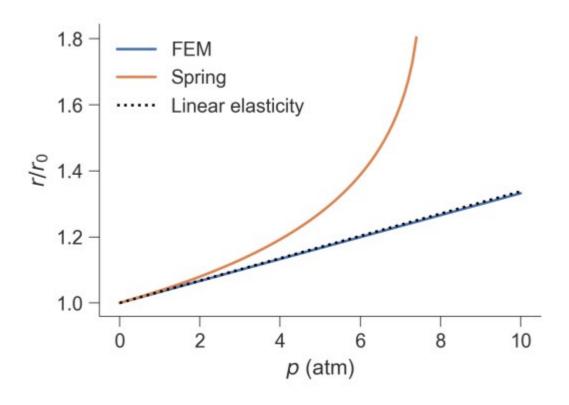
$$\varepsilon_{\text{direction}}(\vec{n}^{(i)}) = \varepsilon_{kl} n_k^{(i)} n_l^{(i)},$$

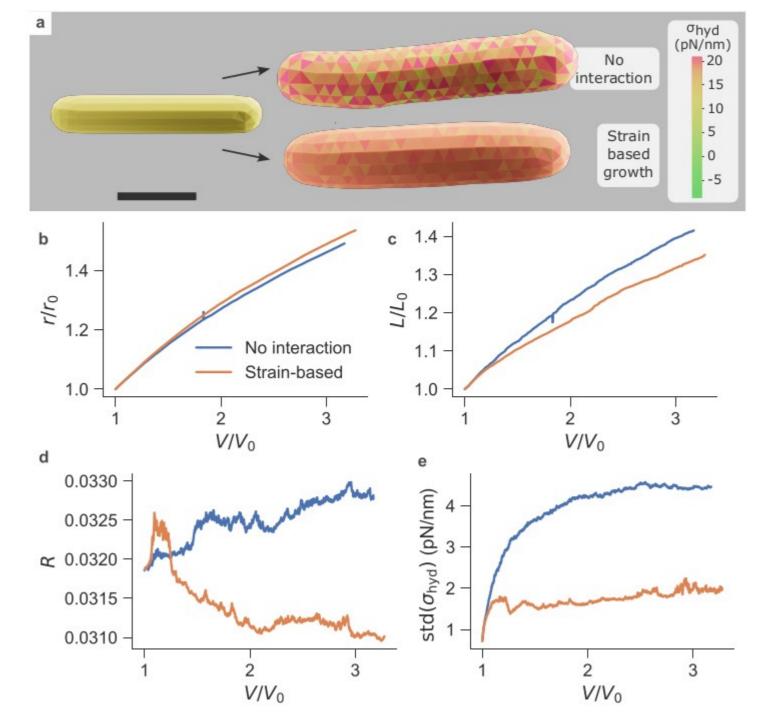
$$\begin{bmatrix} \varepsilon_{\mathrm{direction}}(\vec{n}^{(1)}) \\ \varepsilon_{\mathrm{direction}}(\vec{n}^{(2)}) \\ \varepsilon_{\mathrm{direction}}(\vec{n}^{(3)}) \end{bmatrix} = \begin{bmatrix} n_1^{(1)} n_1^{(1)} & n_1^{(1)} n_2^{(1)} & n_2^{(1)} n_2^{(1)} \\ n_1^{(2)} n_1^{(2)} & n_1^{(2)} n_2^{(2)} & n_2^{(2)} n_2^{(2)} \\ n_1^{(3)} n_1^{(3)} & n_1^{(3)} n_2^{(3)} & n_2^{(3)} n_2^{(3)} \end{bmatrix} \cdot \begin{bmatrix} \varepsilon_{11} \\ 2\varepsilon_{12} \\ \varepsilon_{22} \end{bmatrix}$$



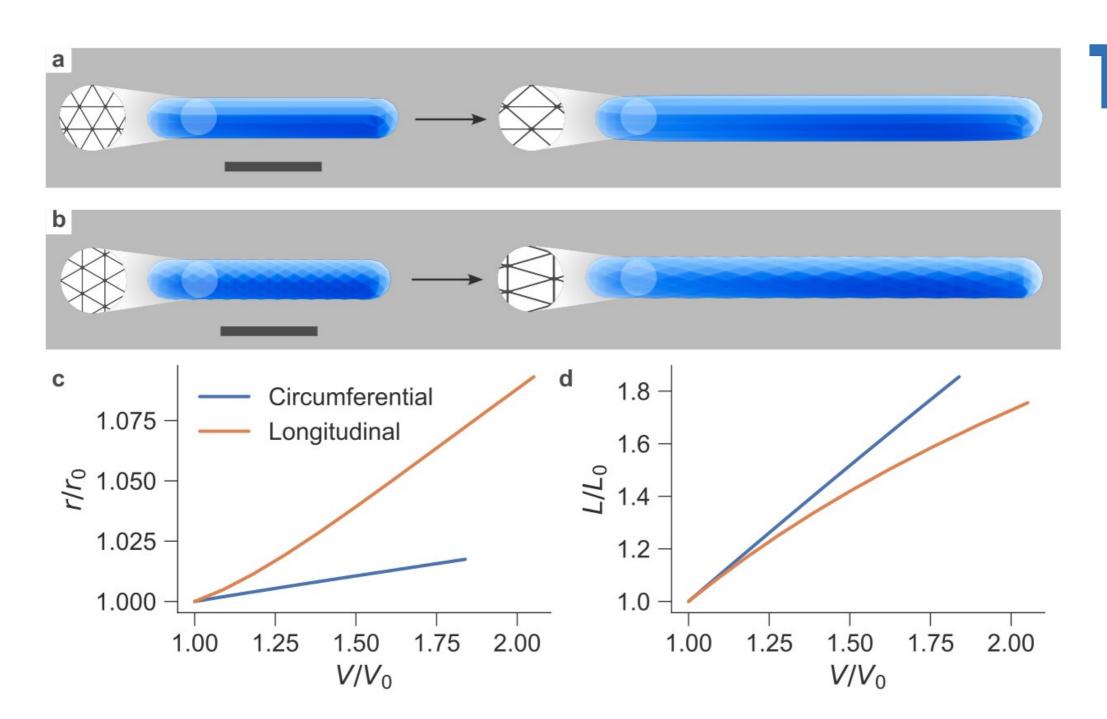


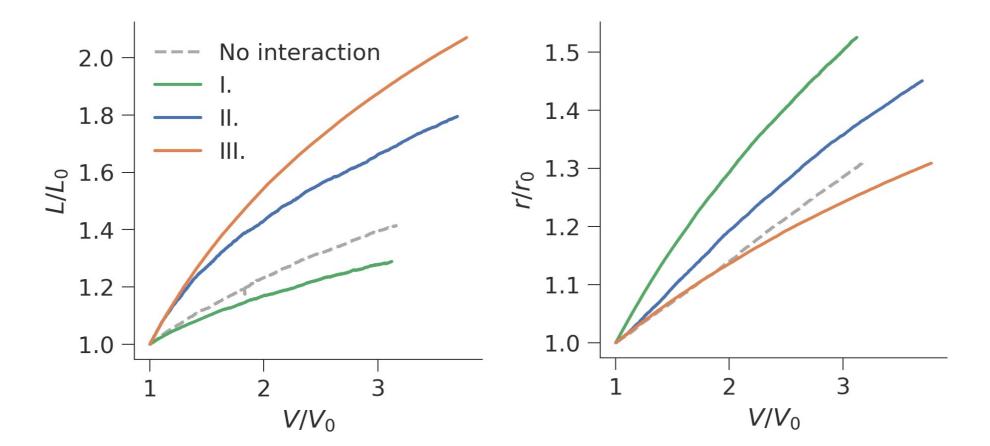










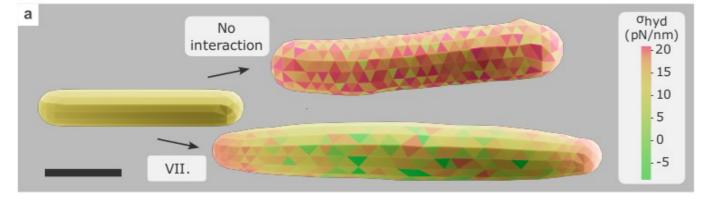




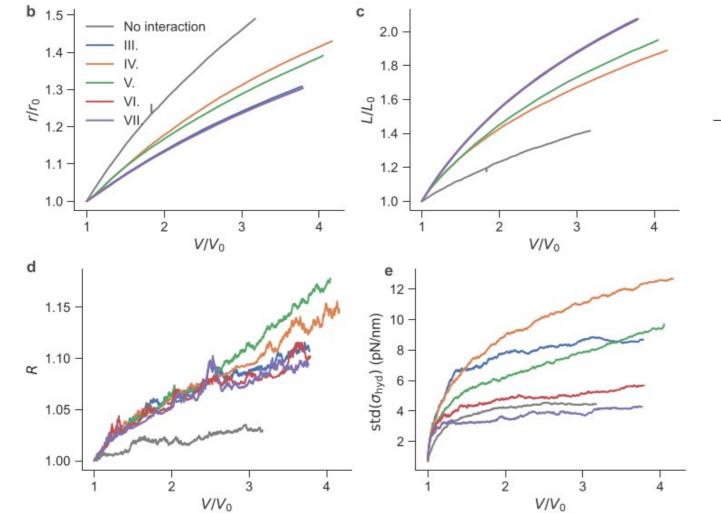
Growth reaction rates

Growth direction

I.
$$\lambda_0 + \lambda_1 \varepsilon_{\min} - \lambda_2 \varepsilon_{\max}$$
 ε_{\min}
II. $\lambda_0 + \lambda_1 K_{\min}$ ε_{\min}
III. $\lambda_0 + \lambda_1 K_{\min}$ K_{\min}







	Growth reaction rates	Growth direction
IV.	$-\lambda_0 + \lambda_1 K_{\min} + \lambda_2 \varepsilon_{\text{hyd}}$	$arepsilon_{ ext{min}}$
V.	$-\lambda_0 + \lambda_1 K_{\min} + \lambda_2 \varepsilon_{\min}$	$arepsilon_{ ext{min}}$
VI.	$-\lambda_0 + \lambda_1 K_{\min} + \lambda_2 \varepsilon_{\text{hyd}}$	K_{\min}
VII.	$-\lambda_0 + \lambda_1 K_{\min} + \lambda_2 \varepsilon_{\min}$	K_{\min}

